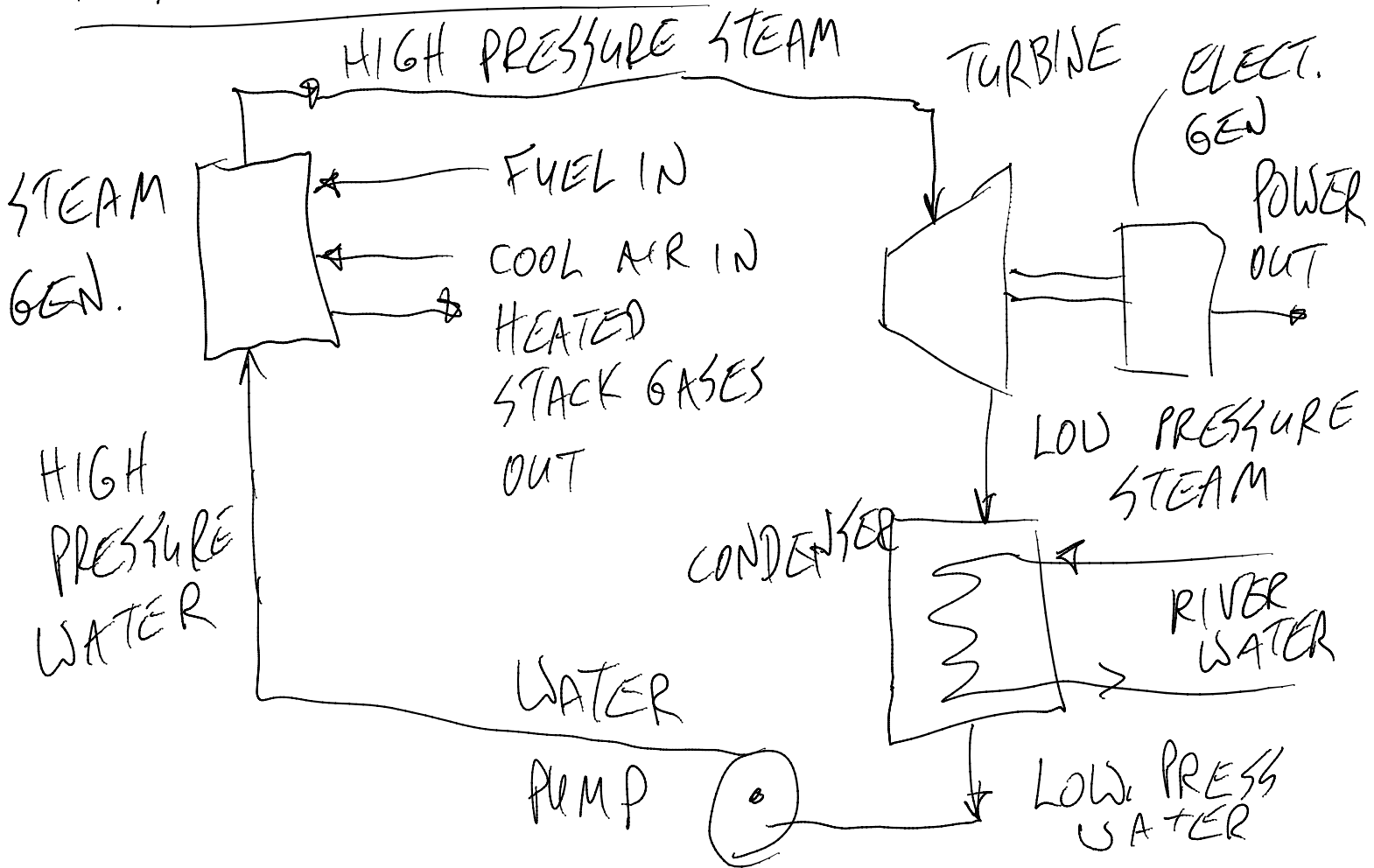


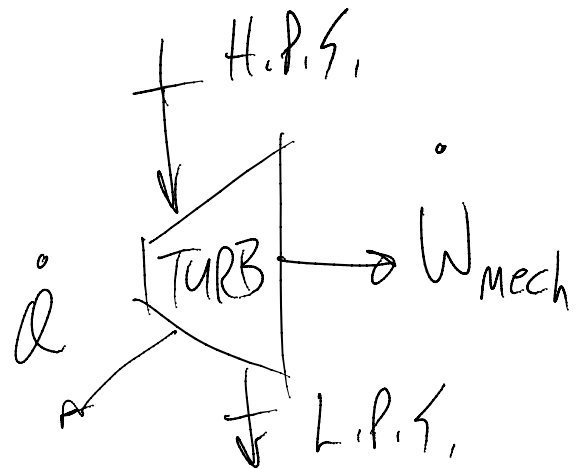
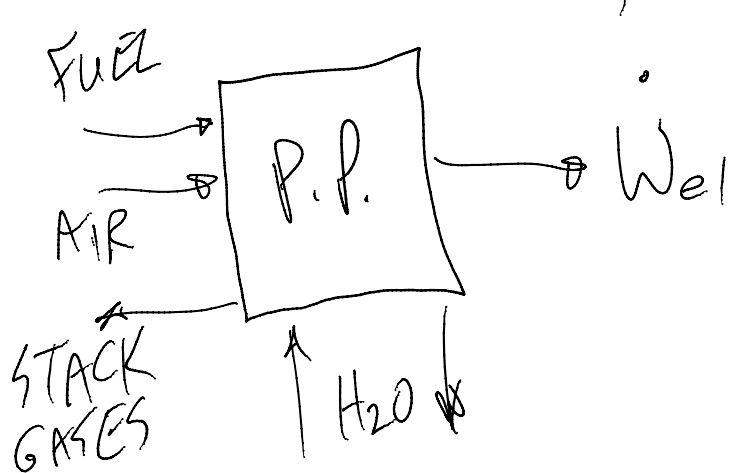
# STEAM POWER PLANT



- PHASE CHANGE (LIQUID/VAPOR, VAPOR/LIQUID)
- PRESSURE/TEMPERATURE/DENSITY CHANGES

• COMBUSTION • HEAT TRANSFER • FLUID FLOW

ANALYZE: GLOBAL, INDIVIDUAL COMPONENTS



OBJECTIVES: FOR A GIVEN POWER OUTPUT, FIND:

• FUEL RATE • COMBUSTION AIR • PUMP POWER

• EFFICIENCY =  $\frac{\text{POWER OUT}}{\text{POWER IN}} < 1$

- PROBLEM SOLVING:

- DEFINE KNOWNS, UNKNOWN

- MAKE UNIT CONVERSIONS

- APPLY LAWS

- SOLVE ALGEBRAICALLY (NO NUMBERS HERE)

$$\dot{Q} = \dot{m} c_p \Delta T$$

$$200 \text{ W} = 35 \cdot 27.9 \cdot \Delta T$$

$$\Delta T = \frac{\dot{Q}}{\dot{m} c_p} = \frac{(200 \text{ W}) \left( \frac{\cancel{\text{J/s}}}{\cancel{\text{W}}} \right) \frac{1}{1000}}{(35 \frac{\cancel{\text{kg}}}{\cancel{\text{s}}}) (4.18 \frac{\cancel{\text{J}}}{\cancel{\text{kg}} \cdot \cancel{\text{K}}})}$$

- SANEITY CHECK ANSWER  $0.000000035$   
3.5

SI VS. ENGLISH

$$F = ma$$

$$(1 \text{ LBF}) = (1 \text{ LBM}) \left( 32.2 \frac{\text{ft}}{\text{s}^2} \right)$$

$$\frac{1 \text{ LBF} \cdot \text{s}^2}{\text{ft}}$$

$$32.2 \text{ LBM} \cdot \text{ft}$$

$$1 \text{ N} = 1 \text{ kg} \cdot 1 \frac{\text{m}}{\text{s}^2}$$

= CONVERSION FOR ENGLISH

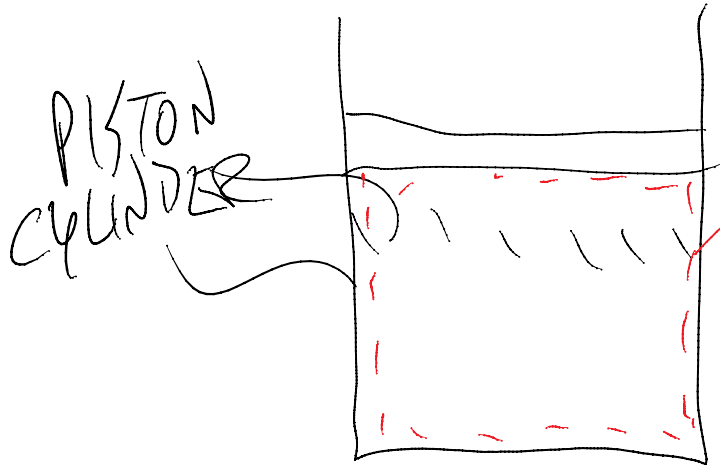
32.2  $\frac{\text{LBM}}{\text{slug}}$

$$CF = \frac{1 \text{ N} \cdot \text{s}^2}{\text{kg} \cdot \text{m}}$$

# DEFINITIONS

- A ~~THERMODYNAMIC~~ SYSTEM IS A QUANTITY OF MATTER OF FIXED MASS AND IDENTITY UPON WHICH ATTENTION IS FOCUSED FOR STUDY.
- THE ~~SURROUNDINGS~~ IS EVERYTHING EXTERNAL TO THE SYSTEM.
- THE SYSTEM BOUNDARIES SEPARATE THE SYSTEM FROM THE SURROUNDINGS, MAY BE FIXED OR MOVABLE.

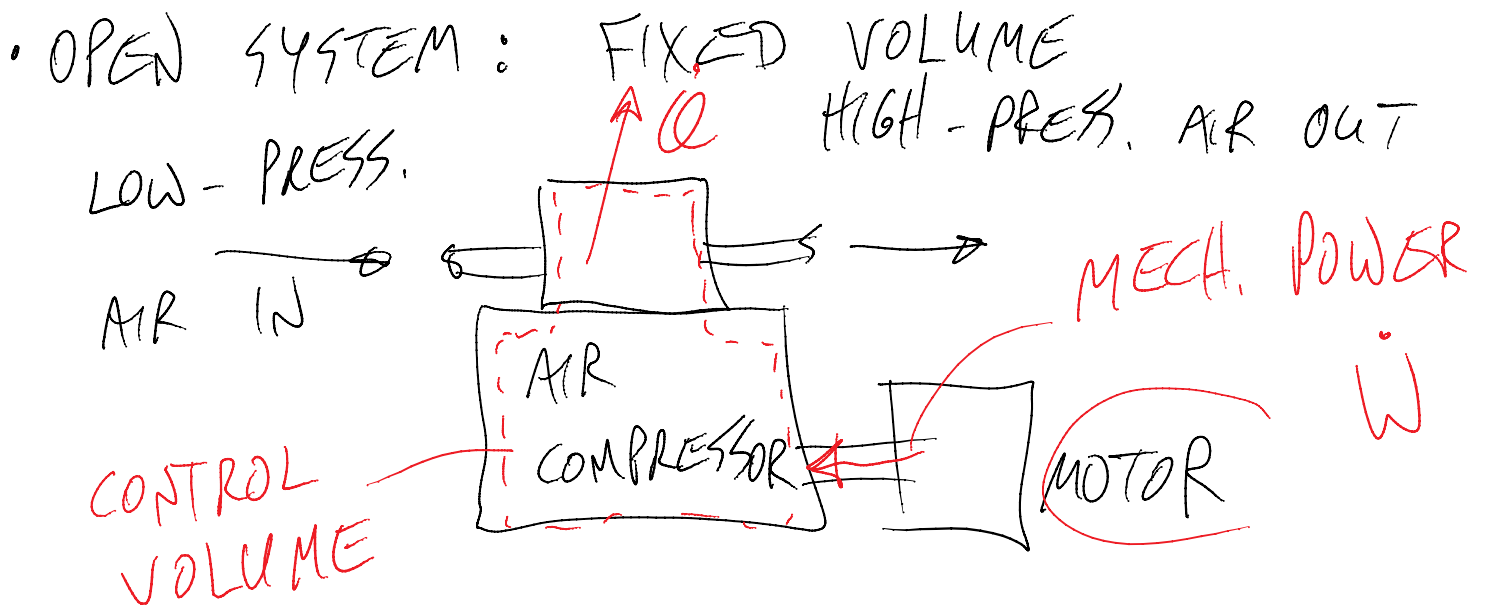
- CLOSED SYSTEM: FIXED QUANTITY OF MASS



SYSTEM BOUNDARIES  
HEAT AND WORK CAN  
CROSS BOUNDARIES

- OPEN SYSTEM: FIXED VOLUME





MASS, MOMENTUM, HEAT AND WORK CROSS THE SYSTEM BOUNDARIES.

- A ~~THERMODYNAMIC STATE~~ OF A SUBSTANCE IS DEFINED BY OBSERVABLE PROPERTIES, SUCH AS PRESSURE, TEMPERATURE, DENSITY,...
- AN ~~EXTENSIVE PROPERTY~~ DEPENDS DIRECTLY ON THE MASS OF THE SYSTEM, SUCH AS VOLUME.
- AN ~~INTENSIVE PROPERTY~~ IS INDEPENDENT

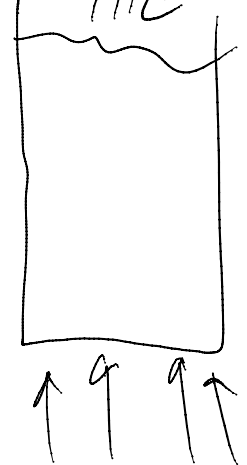
OF MASS, SUCH AS TEMP., PRESS., DENSITY...

AN EXTENSIVE PROPERTY CAN BE CHANGED TO A ~~SPECIFIC PROPERTY~~ BY DIVIDING BY THE SYSTEM MASS.

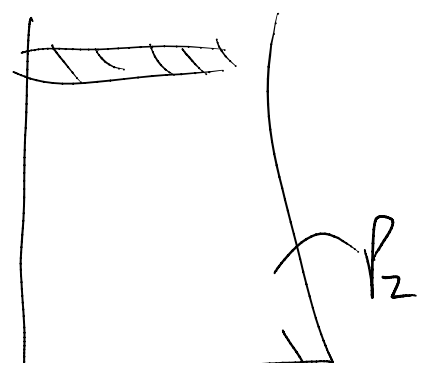
$$\text{SPECIFIC VOLUME} = v = \frac{V}{m} \quad \frac{\text{m}^3}{\text{kg}}, \frac{\text{ft}^3}{\text{LBM}}$$

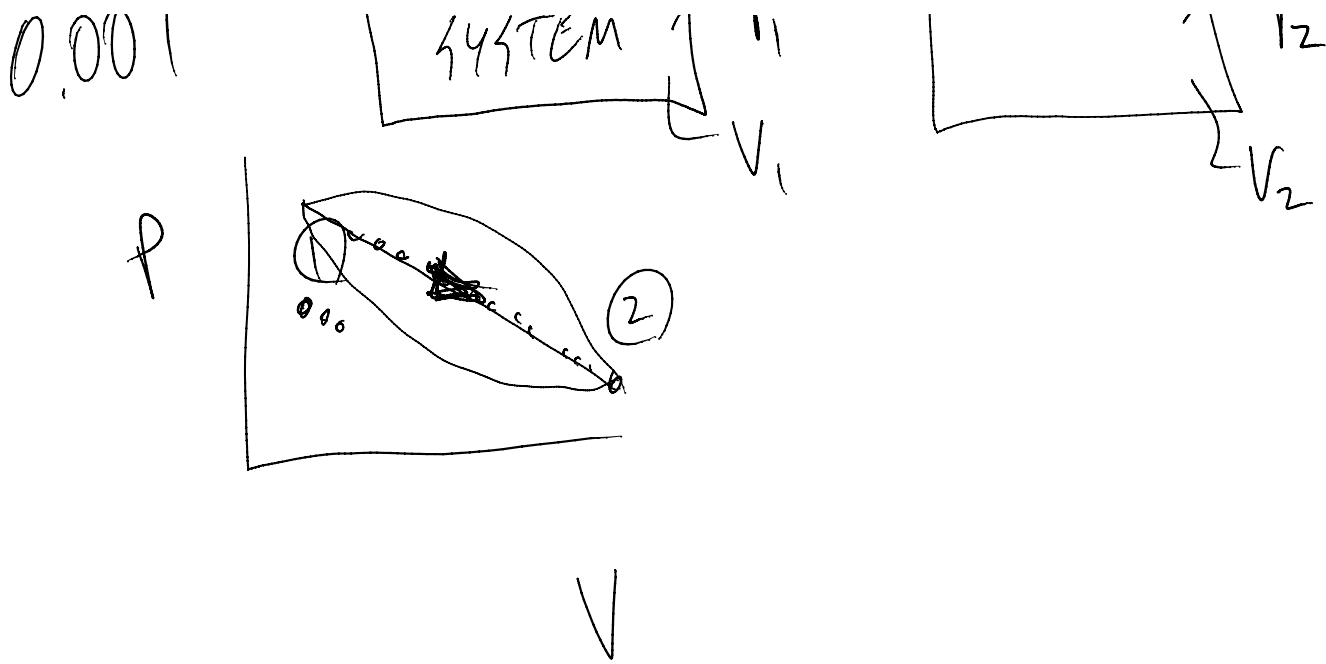
• ~~EQUILIBRIUM~~ MEANS THAT A PARTICULAR PROPERTY IS UNIFORM THROUGHOUT THE SYSTEM.

• A ~~PROCESS~~ IS THE PATH OF THE SUCCESSION OF STATES THROUGH WHICH THE SYSTEM PASSES.



0.001 in





A QUASI-EQUILIBRIUM PROCESS IS ONE IN WHICH THE DEVIATION FROM THERMODYNAMIC EQUILIBRIUM IS INFINITESIMAL.

~~PROB. 31~~

$$\text{MASS} = 3.75 \times 10^9 \frac{\text{kg}}{\text{yr}}$$

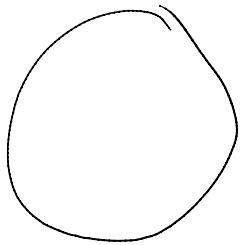
• THE CONTINUUM APPROXIMATION ASSUMES THAT THE MEAN FREE PATH OF THE MOLECULES IS SMALL COMPARED TO VOLUME IN

QUESTION.

~~DEFINITION OF PRESSURE~~

$$P = \lim_{\Delta A \rightarrow a} \frac{\Delta F}{\Delta A}$$

$\Delta F \rightarrow$  FORCE  
 $a \rightarrow$  CONTINUUM APPROXIMATION  
 $\Delta A \rightarrow$  AREA

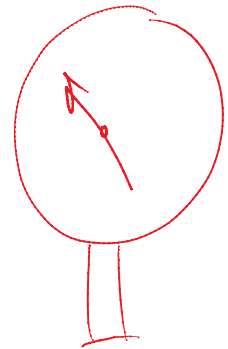
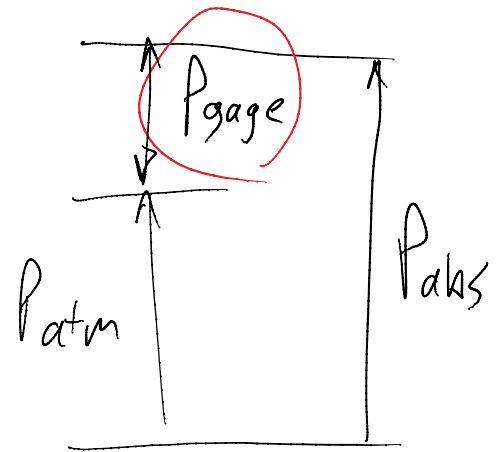


$P_{abs}$  = ABSOLUTE PRESS.

$P_{atm}$  = ATMOSPHERIC PRESS.

$P_{gage}$  = GAGE PRESSURE.

$$P_{gage} = P_{abs} - P_{atm}$$



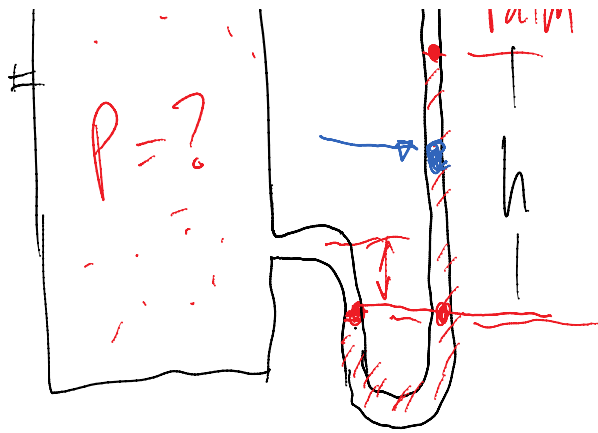
IDEAL GAS RELATION: ABSOLUTE PRESSURE

EXAMPLE

MERCURY <sup>RHO</sup> MANOMETER



$$\rho_m = 13,600 \frac{kg}{m^3}$$

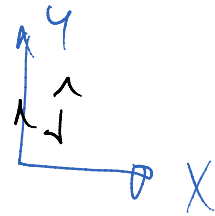
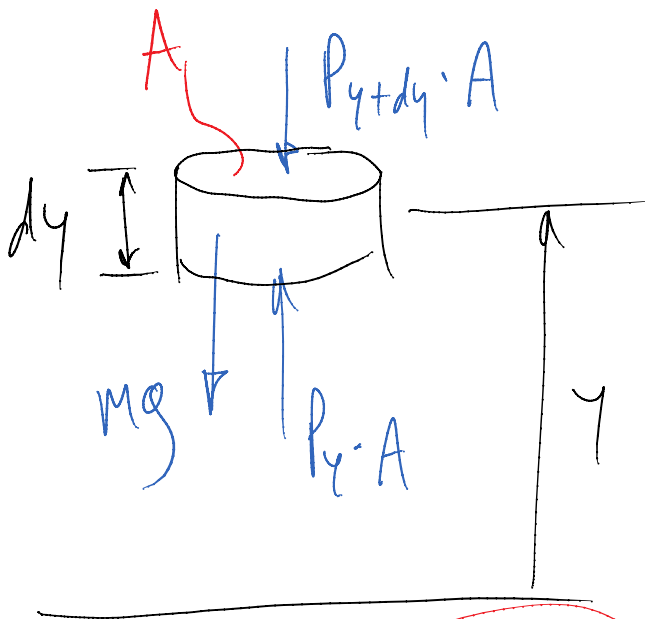


$$\rho_m = 13,550 \frac{\text{kg}}{\text{m}^3}$$

$$h = 2 \text{ m}, \quad g = 9.81 \frac{\text{m}}{\text{s}^2}$$

FIND  $P_{abs}$ ,  $P_{gage}$ . WHAT ABOUT WATER?

( $\rho_w = 1000 \frac{\text{kg}}{\text{m}^3}$ ) FIND  $h$ .



FORCE BALANCE:

$$P_y \cdot A - P_{y+dy} \cdot A - mg = 0$$

$$m = \rho \cdot V = \rho \cdot dy \cdot A$$

$$P_y \cdot A - P_{y+dy} \cdot A - \rho g A dy = 0$$

0 0 . e . l . - a

→

→ 0

$$P_y - P_{y+dy} - \rho g dy = 0$$

$$P_{y+dy} = P_y + \left(\frac{dP_y}{dy}\right) \cdot dy + \left(\frac{d^2 P_y}{dy^2}\right) \cdot \left(\frac{dy^2}{2!}\right) + \dots$$

TAYLOR SERIES: NEGLECT HIGHER-ORDER TERMS

$$P_y - \left[ P_y + \left(\frac{dP_y}{dy}\right) \cdot dy \right] - \rho g dy = 0$$

$$\frac{dP_y}{dy} = -\rho g$$

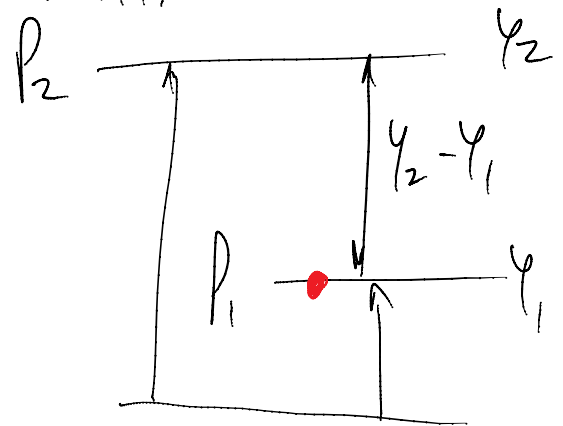
SEPARATE VARIABLES + INTEGRATING:

$$dP_y = (-\rho g) dy$$

$$\int_1^2 dP_y = (-\rho g) \int_1^2 dy$$

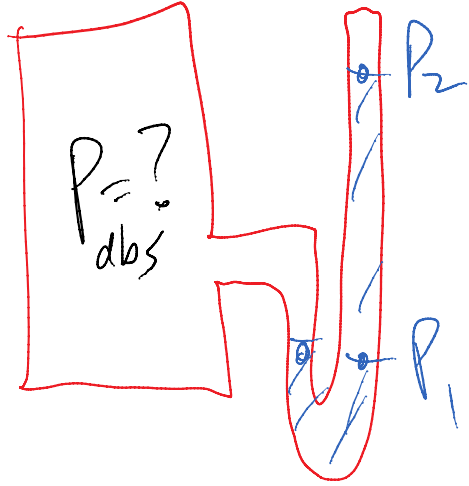
$$P_2 - P_1 = (-\rho g)(y_2 - y_1)$$

$$\text{LET } y_2 - y_1 = h$$



$$P_1 = P_2 + \rho g h$$

$$P_{abs} = P_{atm} + \rho g h$$



FROM BEFORE,

$$P_{gage} = P_{abs} - P_{atm} = \rho g h$$

$$P_{atm} = 1.01 \times 10^5 \frac{N}{m^2}$$

$$\frac{N}{m^2} = Pa$$

$$P_{abs} = \left(1.01 \times 10^5 \frac{N}{m^2}\right) + \left(13,550 \frac{kg}{m^3}\right) \left(9.81 \frac{m}{s^2}\right) (2.0 m) \left(\frac{1 N \cdot s^2}{kg \cdot m}\right)$$

$$P_{atm} = 100 \text{ kPa} \quad 1 \text{ kPa} = 1000 \text{ Pa} = 1000 \frac{N}{m^2}$$

$$P_{abs} = 3.66 \times 10^5 \frac{N}{m^2}$$

$$P_{gage} = \left(3.66 \times 10^5 \frac{N}{m^2}\right) - \left(1.01 \times 10^5 \frac{N}{m^2}\right)$$

$$P_{gage} = 2.65 \times 10^5 \frac{N}{m^2}$$

FIND  $h$  IF FLUID IS WATER:

$$P_{\text{gage}} = \rho g h$$

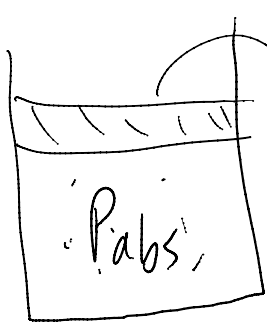
$$\checkmark F = ma$$

$$h = \frac{P_{\text{gage}}}{\rho_w g} = \frac{\left(2.65 \times 10^5 \frac{\text{N}}{\text{m}^2}\right) \left(1.0 \frac{\text{kg} \cdot \text{m}}{\text{N} \cdot \text{s}^2}\right)}{\left(1000 \frac{\text{kg}}{\text{m}^3}\right) \left(9.81 \frac{\text{m}}{\text{s}^2}\right)}$$

$$h = 27.0 \text{ m}$$

### EXAMPLE

DREADED PISTON/CYLINDER ARRANGEMENT:



$$A = 10 \text{ ft}^2$$

$$g = 32.2 \frac{\text{ft}}{\text{s}^2}$$

$$P_{\text{atm}} = 14.7 \text{ psia}, \quad P_{\text{abs}} = 30 \text{ psia}$$

$$\text{psia} = \left( \frac{\text{LBF}}{\text{IN}^2} \right)_{\text{ABSOLUTE}}$$

FIND THE MASS OF THE PISTON.

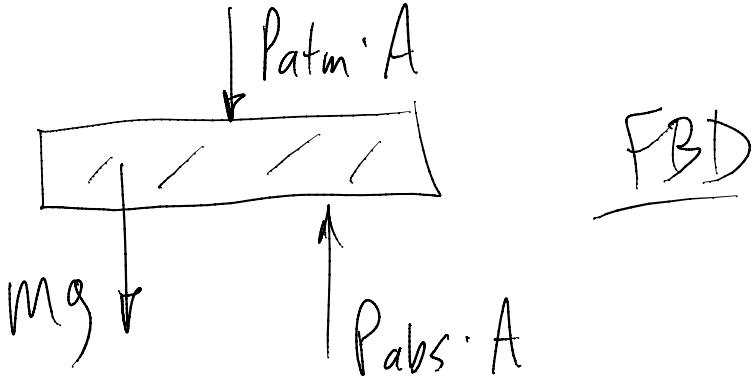
$$P_{\text{atm}} = \left( 14.7 \frac{\text{LBF}}{\text{IN}^2} \right) \left( \frac{12 \text{ IN}^2}{\text{ft}^2} \right) = 2117 \frac{\text{LBF}}{\text{ft}^2}$$

$$D. \dots = 4220 \text{ LBF}$$



$$P_{abs} = 4320 \frac{\text{LBF}}{\text{ft}^2}$$

$$P = \frac{F}{A}$$



$$P_{abs} \cdot A = P_{atm} \cdot A + mg$$

$$F = ma$$

$$1 \text{ LBF} = (1 \text{ LBM}) \left( 32.2 \frac{\text{ft}}{\text{s}^2} \right)$$

$$m = \frac{A}{g} (P_{abs} - P_{atm})$$

$$m = \frac{(10 \text{ ft}^2)}{(32.2 \frac{\text{ft}}{\text{s}^2})} \cdot \left( 4320 - 2117 \frac{\text{LBF}}{\text{ft}^2} \right) \left( \frac{32.2 \text{ LBM} \cdot \text{ft}}{\text{LBF} \cdot \text{s}^2} \right)$$

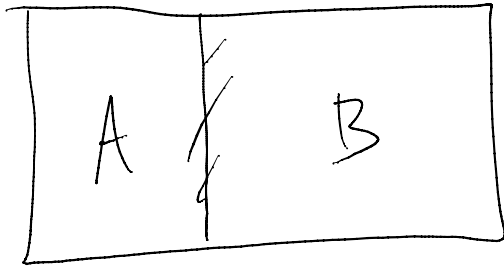
$$M = 2.2 \times 10^4 \text{ LBM}$$

DEFINITION OF SPECIFIC VOLUME:

$$v = \frac{V}{M} = \frac{\text{SYSTEM VOLUME}}{\text{SYSTEM MASS}}$$

$$U = \frac{1}{\rho}$$

EXAMPLE

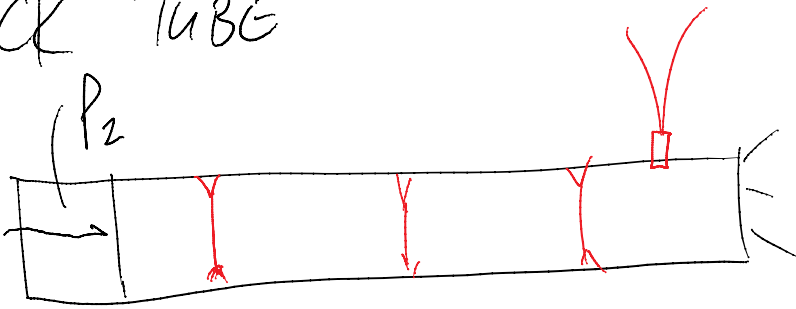


$$U_A = 17.196 \frac{\text{m}^3}{\text{kg}}, \quad V_A = 0.2 \text{m}^3$$

$$U_B = 3.418 \frac{\text{m}^3}{\text{kg}}, \quad V_B = 0.5 \text{m}^3$$

FIND  $U_T$

SHOCK TUBE



$$U_A = \frac{V_A}{M_A}, \quad U_B = \frac{V_B}{M_B}$$

CONSERVE MASS:  $M_T = M_A + M_B$

TOTAL VOLUME:  $V_T = V_A + V_B$

TOTAL SPECIFIC VOLUME:

$$U_T = \frac{V_T}{M_T} = \frac{V_A + V_B}{M_A + M_B}, \quad M_A = \frac{V_A}{U_A}, \quad M_B = \frac{V_B}{U_B}$$

$$U_T = \frac{V_T}{M_T} = \frac{V_A + V_B}{M_A + M_B}, \quad M_A = \frac{V_A}{U_A}, \quad M_B = \frac{V_B}{U_B}$$

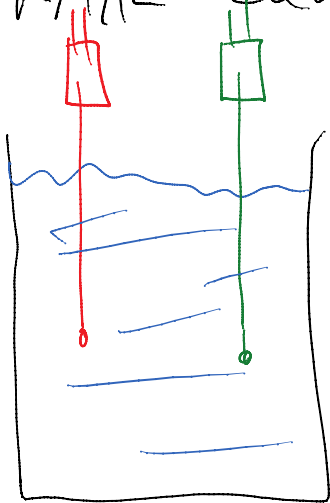
$$U_T = \frac{V_A + V_B}{V_A/U_A + V_B/U_B}$$

$$U_T = \frac{(0.2 \text{ m}^3) + (0.5 \text{ m}^3)}{(0.2 \text{ m}^3) / (17.196 \frac{\text{m}^3}{\text{kg}}) + (0.5 \text{ m}^3) / (3.418 \frac{\text{m}^3}{\text{kg}})}$$

$$U_T = 4.43 \frac{\text{m}^3}{\text{kg}}$$

## ZEROTH LAW OF THERMO

TWO BODIES WHICH ARE EACH IN THERMAL EQUILIBRIUM WITH A THIRD BODY ARE IN THERMAL EQUILIBRIUM WITH EACH OTHER.



THEMOCOUPLE

RESISTANCE TEMPERATURE  
DETECTOR

WATER:  $\pm 0.015^\circ\text{C}$

→ calibration

WATER - 100°C -

COST

\$28.00

\$7000.00

ACCURACY

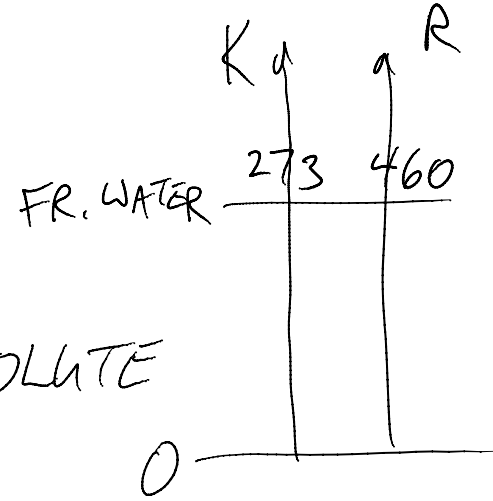
$\pm 1.0^{\circ}\text{C}$

$\pm 0.0043^{\circ}\text{C}$

CALIBRATE

$\pm 0.023^{\circ}\text{C}$

$$\dot{Q} = \dot{m} C_p (T_o - T_i)$$



## TEMPERATURE

KELVIN, RANKINE ARE ABSOLUTE TEMPERATURE SCALES:

$$1.8^{\circ}\text{R} = 1\text{K}$$

CELSIUS, FAHRENHEIT:

$$T(^{\circ}\text{C}) = T(\text{K}) - 273, \quad T(^{\circ}\text{F}) = T(^{\circ}\text{R}) - 460$$

$$T(^{\circ}\text{F}) = 1.8 \cdot T(^{\circ}\text{C}) + 32$$

# Pvt diagram

Thursday, May 09, 2013  
12:53 PM

